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Model-free Load Control for High Penetration of Solar Photovoltaic Generation

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Abstract—This paper presents a new model-free control (MFC) mechanism that enables the local distribution level circuit consumption of the photovoltaic (PV) generation by local building loads, in particular, distributed heating, ventilation, and air conditioning (HVAC) units. The local consumption of PV generation will help minimize the impact of PV generation on the distribution grid, reduce the required battery storage capacity for PV penetration, and increase solar PV generation penetration levels. The proposed MFC approach with its corresponding intelligent controllers does not require any precise model for buildings, where a reliable modeling is a demanding task. Even when assuming the availability of a good model, the various building architectures would compromise the performance objectives of any model-based control strategy. The objective is to consume most of the PV generation locally while maintaining occupants comfort and physical constraints of HVAC units. That is, by enabling proper scheduling of responsive loads temporally and spatially to minimize the difference between demand and PV production, it would be possible to reduce voltage variations and two-way power flow. Computer simulations show promising results where a significant proportion of the PV generation can be consumed by building HVAC units with the help of intelligent control.

Index Terms—HVAC, distributed energy resources, building load control, solar variability, model-free control, intelligent proportional controllers.

I. INTRODUCTION

A large introduction of solar photovoltaic (PV) and other renewable resources may significantly alter the stability and balance of the power grid, through voltage fluctuations and the ability to maintain the required network frequency [1]. A viable solution to this problem, knowing the widespread use of HVAC systems in residential and commercial buildings, is to use the idea of load shaping. By introducing responsive loads (buildings), the control of HVAC units will act as a counter to undesirable fluctuations of power generated by the building sized PV arrays, while not sacrificing some level of temperature comfort.

The motivation of not relying on model-based control approaches comes from the daunting task of devising a reliable building model, either via physical laws or via blackbox identification, and also, in the fact that it is difficult to account for all the uncertainties and unknown disturbances, especially with respect to strong weather disturbances and changes in occupancy pattern (see, *e.g.*, [2], [3], [4], [5], [6] and the references therein). A good model often requires the installation of too many sensors necessary for the control algorithm decision making part. Not to mention the financial cost that this will incur.

We are therefore proposing to use a new model-free control (MFC) approach with its corresponding intelligent controllers [7], where the need of any precise modeling disappears. This setting

- is *data-driven*, *i.e.*, only the input and output data are used. Mathematical physical laws and the associated differential equations are ignored.
- is not limited to finite-dimensional linear systems. It has been successfully applied in much more involved situations with nonlinearities and where modeling via partial differential equations has been proposed.¹
- is easy to implement and quite robust with respect to internal and external disturbances, even when compared to the most popular Proportional Integral Derivative controller (see, *e.g.*, [9]);
- has already been successfully employed all over the world and gave rise to some patents: see, *e.g.*, the references in [7], [10] and the references therein, and [11], [12], [13], [14], [15], [16], [17]. For obvious reasons let us emphasize here the heating and the humidification of an agricultural greenhouse [18], and the heating of a single building [19].

Although the idea of using HVAC unit systems to provide ancillary services to the grid is not new (see [20], [21]), the main contribution of this paper is on designing a MFC framework in order to consume most of the generated PV power locally by building loads, while simultaneously ensuring a certain temperature comfort inside the buildings. How

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¹Employing PDEs is quite common in the air conditioning of buildings. See, e.g., [8] and the references therein.

to solve the actual problem turns out to be intuitive, in the sense that one can consider the total energy provided by the PV profile and share it amongst the available building load. A straightforward solution was then to consider at each instant of time the available energy coming from the profile as a constraint for each HVAC system. Note that the difficulty in trying to implement such procedures is the necessity for the control to follow two reference trajectories simultaneously (see also [22]). One corresponding to the interior temperature of the buildings (set in advance within a certain comfort zone), and the other, corresponding to the generated PV profile. It is clearly a hard requirement to satisfy if one wants to avoid optimization problem, where it is easier to introduce those constraints. The price to pay for such optimization will result in higher computational power, even if sub-optimality mathematical techniques are introduced (see, e.g., [8], [23], [24], [25] and the references therein). A real-time implementation would thus become problematic.

The remainder of the paper is organized as follows. Section II provides a short introduction to model-free control control and to the corresponding intelligent controllers.² A simplified physical model is presented in Section III together with the formulation of our approach and the simulation results. Section IV summarizes the paper and presents the conclusions.

II. MODEL-FREE CONTROL AND INTELLIGENT CONTROLLERS

Without any loss of generality we restrict ourselves to Single-Input Single-Output systems. Instead of trying to write down a complex differential equation, introduce the *ultra-local* model

$$\dot{y} = F + \alpha u \tag{1}$$

where

- u and y are the input (control) and output variables,
- the derivation order of y is 1, like in most concrete situations,
- $\alpha \in \mathbb{R}$ is chosen by the practitioner such that αu and \dot{y} are of the same magnitude.

The following explanations on F might be useful:

- F subsumes the knowledge of any model uncertainties and disturbances,
- F is estimated via the measures of u and y.

A. Intelligent controllers

The loop is closed by an *intelligent proportional controller*, or *iP*,

$$u = -\frac{\ddot{F} - \dot{y}^* + K_P e}{\alpha} \tag{2}$$

where

- y^* is the reference trajectory,
- $e = y y^*$ is the tracking error,
- K_P is the usual tuning gain.

²See [7] for full details.

Combining equations (1) and (2) yields:

$$\dot{e} + K_P e = 0$$

where F does not appear anymore. Local exponential stability is ensured if Kp > 0:

- The gain K_P is thus easily tuned.
- Robustness with respect to different types of disturbances and model uncertainties is achieved.

B. Estimation of F

F is estimated in real-time according to recent algebraic identification techniques [26], [27], [28].

1) First approach: The term F in Equation (1) may be assumed to be "well" approximated by a piecewise constant function \hat{F} . Rewrite then Equation (1) in the operational domain (see, *e.g.*, [29]):

$$sY = \frac{\Phi}{s} + \alpha U + y(0) \tag{3}$$

where Φ is a constant. We get rid of the initial condition y(0) by multiplying both sides on the left by $\frac{d}{ds}$:

$$Y + s\frac{dY}{ds} = -\frac{\Phi}{s^2} + \alpha \frac{dU}{ds} \tag{4}$$

Noise attenuation is achieved by multiplying both sides on the left by s^{-2} . It yields in the time domain the real-time estimate, thanks to the equivalence between $\frac{d}{ds}$ and the multiplication by -t,

$$\hat{F}(t) = -\frac{6}{\tau^3} \int_{t-\tau}^t \left[(\tau - 2\sigma)y(\sigma) + \alpha\sigma(\tau - \sigma)u(\sigma) \right] d\sigma$$
(5)

2) Second approach: Close the loop with the iP (2):

$$\hat{F}(t) = \frac{1}{\tau} \left[\int_{t-\tau}^{t} \left(\dot{y}^{\star} - \alpha u - K_{P} e \right) d\sigma \right]$$
(6)

Remark 2.1: Note the following facts:

- Integrals (5) and (6) are low pass filters.
- $\tau > 0$ may be chosen quite small.
- Integrals (5) and (6) may of course be replaced in practice by classic digital filters.

III. PROBLEM FORMULATION AND COMPUTER SIMULATIONS

A. A simple mathematical model

Computer simulations require obviously some mathematical modeling. We have selected a simple set of linear differential equations, which are time-invariant, *i.e.*, with constant coefficients. It was derived in [30] and successfully used in [23], [31], [32]. The dynamic of the interior temperature, interior

wall surface temperature, and exterior wall core temperature are given by

$$\dot{T}_{1} = \frac{1}{C_{1}} \left[(K_{1} + K_{2})(T_{2} - T_{1}) + K_{5}(T_{3} - T_{1}) + u_{c} + \delta_{2} + \delta_{3} \right]$$

$$\dot{T}_{2} = \frac{1}{C_{2}} \left[(K_{1} + K_{2})(T_{1} - T_{2}) + \delta_{2} \right]$$

$$\dot{T}_{3} = \frac{1}{C_{3}} \left[K_{5}(T_{1} - T_{3}) + K_{4}(\delta_{1} - T_{3}) \right]$$
(7)

$$\dot{x} = A_c x + B_c u + C_c w \tag{8}$$

where the state vector x, the control u and the disturbance w are given respectively as

$$x = \begin{bmatrix} T_1 \\ T_2 \\ T_3 \end{bmatrix}, \quad u = u_c, \quad w = \begin{bmatrix} \delta_1 \\ \delta_2 \\ \delta_3 \end{bmatrix}$$

 T_1 : room air temperature [°C] $22 \le T_1 \le 24$,

 T_2 : interior-wall surface temperature [°C],

 T_3 : exterior-wall core temperature [°C],

 u_c : cooling power (≤ 0) [kW],

 δ_1 : outside air temperature [°C],

- δ_2 : solar radiation $[kW/m^2]$,
- δ_3 : internal heat sources [kW],

 $C_1 = 9.356 \times 10^5$ kJ/C, $C_2 = 2.970 \times 10^6$ kJ/C, $C_3 = 6.695 \times 10^5$ kJ/C, $K_1 = 16.48$ kW/C, $K_2 = 108.5$ kW/C, $K_3 = 5$, $K_4 = 30.5$ kW/C and $K_5 = 23.04$ kW/C.

The above equations are only valid for a summer day. The main control variable is therefore cooling power u_c . One of our control objectives is to regulate the interior temperature of several buildings around 23 °C. This is the reference trajectory.

Remark 3.1: A poor knowledge of the coefficients in Equation (8) is unavoidable in practice. Lack of space prevents us of confirming via simulations the robustness of our MFC setting in such a situation.

B. Temperature regulation and PV following

Our aim is to regulate the interior temperature of multiple buildings while at the same time assuring the whole consumed energy tracks as closely as possible the PV energy profile. It yields the following control objectives:

- Maintain the interior temperature under an acceptable comfort zone, *i.e.*, between 22°C and 24°C.
- Follow the generated PV profile as closely as possible.

Remark 3.2: The PV profile is the only energy that the HVAC units are supposed to get. The buildings are also connected to the grid. It means that the total energy consumed is allowed to fluctuate around the generated PV signal.

Instead of considering the PV profile as a constraint, as it has been mentioned in the introduction, *i.e.*,

$$\sum_{i=1}^{N_b} u_i(t) \le PV(t) \tag{9}$$

where N_b is the number of buildings, a better alternative is to assume a band around the PV profile, such that the available energy is given according to

$$E_d(t): \begin{cases} E_d(t) = 0 & \text{if } PV(t) = 0\\ PV(t) - \epsilon \le E_d(t) \le PV(t) + \epsilon & \text{if } PV(t) > 0 \end{cases}$$

where

- *ϵ* > 0 is a constant, which is chosen according to the grid capabilities and the comfort resulting from the interior temperature;
- it is assumed that, when $E_d(t) = 0$, all the buildings are connected to the grid and no PV tracking is performed.
- The constraint 0 ≤ u_c ≤ 3 on the cooling power must be satisfied when there is no PV energy.

Therefore the total energy consumption is

$$PV(t) - \epsilon \le \sum_{i=1}^{N_b} u_i(t) \le PV(t) + \epsilon$$
(10)

Assume for simplicity's sake that all the buildings are identical. Then Inequality (10) yields

$$\frac{PV(t)}{N_b} - \frac{\epsilon}{N_b} \le u_i(t) \le \frac{PV(t)}{N_b} + \frac{\epsilon}{N_b}, \quad i = 1, 2, ..N_b$$
(11)

C. Computer simulations

A 10 minute sampling is used.³ Consider first 13 buildings. The simulations are presented first when only the interior temperatures T_{1i} , $i = 1, ..., N_b$, is regulated. Afterwards a second objective is added, *i.e.*, the tracking of the PV generated signal, *i.e.*, Inequality (10) must be satisfied.

1) Control design: Since the interior temperatures T_{1i} are available for measurements, Equation (1) yields the ultra-local model

$$\dot{T}_{1i} = F_i + \alpha_i u_{ci} \tag{12}$$

From Equation (2) we deduce the following intelligent proportional controller

$$u_{ci} = -\frac{\hat{F}_i - \dot{T}_{1i}^* + K_{Pi}e_{p_i}}{\alpha_i}$$
(13)

where

- $\alpha_i = 5, K_{P_i} = 2,$
- $e_{p_i} = T_{1i} T_{1i}^*$ is the tracking error,
- the reference trajectory T_{1i}^* should be as close as possible to the reference temperature $23^\circ C$,
- \hat{F}_i is an estimate of F, which is obtained according to Section II-B,
- for a "good" PV tracking each control u_{ci} must satisfy Inequality (11).

 3 The time lapse depends heavily, of course, on the nature of the plant we want to regulate.



Figure 1. Interior temperature variations in $^{\circ}C$ for $N_b = 13$



Figure 2. Time evolution of the 13 control inputs

2) Model-free control with no PV tracking: Figure 1 displays the evolution of the interior temperature during three days. The temperature is perfectly regulated. MFC is very efficient in rejecting outside disturbances: one can see their effect in the little bumps appearing essentially from the 8^{th} to the 20^{th} hour of the day. Note also that the initial temperatures in each building are different. The control inputs in Figure 2 satisfy the required constraints. The initial large values correspond to the buildings with the highest initial temperature. It is important to notice in Figure 3 that the total energy used by the HVAC units does not take into account the variations of the PV profile, hence the large deviation of the total energy signal represented in blue dashed lines from the actual PV given in red solid line.

3) Simulation with PV profile: As soon as the photovoltaic source starts collecting energy, Inequality (11) is taken into account. Figure 4 displays the time evolution of the interior



Figure 3. Total energy $\sum_{i=1}^{N_b} u_{ci}(t)$ and the PV profile

temperature for 13 buildings while the PV tracking is taken into account. Note that most of the temperatures are within their prescribed comfort zone. The slight violation of the lower constraints for one or two buildings is due to the tight restriction on the PV tracking as shown by Figure 5: we select $\epsilon = 1$, *i.e.*, a rather narrow tolerance margin. Temperatures where slight overshoots appear are those that are starting from the lowest initial temperatures. By imposing more energy consumption for the buildings than necessary for regulating its indoor temperature, one would expect the HVAC units to run during more time or with higher intensity. It would vield large variations. This behavior is rather normal. It can be significantly improved by selecting an appropriate number of buildings. It is important to understand the difficulty of the actual compromise between the requirement of imposing a certain level of comfort inside the buildings, and that of closely following the generated PV signal. Those two requirements are very often contradictory. Our viewpoint helps in satisfying both control objectives without the need of any complex optimization procedure, especially if one has to deal with a nonlinear model. As an initial study we decided to focus on a limited number of HVAC units to assess our results. However scaling the problem to larger building units is straightforward. It does not necessitate any change in the problem formulation. More importantly it will not require any additional significant computational power.

As already mentioned, it is possible to improve the results in Figure 4. Add for that purpose one building, *i.e.*, $N_b = 14$. Figure 7 shows a clear-cut improvement with respect to the interior temperature. Figures 8 and 9 do not display on the other hand any significant change with respect to the case of 13 buildings. One needs to be cautious when dealing with an unfeasible scenario where the maximum energy consumed by all buildings is very small when compared to the energy coming from the PV source, or when the PV energy is not sufficient to account for all the HVAC units. Thus, a better



Figure 4. Interior temperature variations in $^{\circ}C$ for $N_b = 13$

temperature regulation emphasizes that we are, in fact, in the scenario where more energy was available from the PV for the 13 buildings. Adding one more helped in distributing the extra energy to the new building, putting less weight on the remaining ones to follow the PV profile.

The above results are quite good. Remember that enabling the buildings to absorb the PV sources will help the latter to minimize their negative effect on the distribution grid and consequently to provide responsive loads that would ultimately help reducing voltage variations and two-way power flow.

IV. CONCLUSION

In order to offer a better integration of the photovoltaic energy sources into the grid, a model-free control approach has been proposed. Computer simulations are showing that the responsive loads (HVAC units) may be regulated in such a way that the tracking error between demand and PV generation is minimized. This fact will help reducing voltage variations and two-way power flow. The two main control objectives mentioned in the introduction are thus satisfied:

- the interior temperatures inside each unit remains in the band $\pm 1^{\circ}C$ around the reference temperature,
- the tracking error of the PV profile is less than 1kW.

A forthcoming publication will extend this work to Boolean controls, *i.e.*, on/off controls. This step should lead to concrete implementations. Let us emphasize finally that the hardware implementation of our control strategy is cheap and easy [33].

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Figure 5. Total energy $\sum_{i=1}^{N_b} u_{ci}(t)$ and the PV profile $(N_b = 13)$



Figure 6. Time evolution of the 13 control inputs

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Figure 7. Interior temperature variations in $^{\circ}C$ for $N_b = 14$





Figure 8. Total energy $\sum_{i=1}^{N_b} u_{c\,i}(t)$ and the PV profile $(N_b=14)$

Figure 9. Time evolution of the 14 control inputs

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