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Cédric Join, Hassane Abouaïssa and Michel Fliess

Abstract The aim of ramp metering is to improve the highway traffic conditions by an appropriate regulation of the inflow from the on-ramps to the highway mainstream. Our presentation rests on several improvements: 1) Our simulation techniques do not need contrarily to other approaches any heuristic fundamental law. 2) There is no need of crucial time-varying quantities, like the critical density, which is most difficult to estimate correctly online. 3) Our feedback loop, which is stemming from model-free control, is easy to implement and shows an excellent robustness with respect to model mismatch. Several computer experiments are displayed and discussed.

Key Words Ramp metering, Lighthill-Whitham-Richards partial differential equation, fundamental diagrams, fundamental laws, ALINEA, model-free control, intelligent proportional controllers.

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1 Introduction

Ramp metering is about the use of traffic signals at highway on-ramps in order to control the rate of vehicles entering the highway. The signals can be set for different metering rates to optimize freeway flow and minimize congestion. The social and ecological damages due to traffic jams justify the huge academic literature which has been devoted to traffic flow dynamics and its regulation. See, *e.g.*, [20, 21, 22, 23, 28, 29, 32, 39], and references therein. Lack of space prevents us from a careful analysis of the various standpoints which are often antagonist.

Our contributions may be summarized like follows:

- 1. No fundamental diagram for simulation purposes. Before being implemented in practice any ramp metering procedure ought to be tested via computers. The similarities between highway traffic and hydrodynamics explain the importance of the Lighthill-Whitham-Richards (LWR) first order partial differential equation [26, 37]. Its numerical integration necessitates to know the relationship between the traffic density and the traffic speed. Several empirical laws, called *fundamental diagrams*, have been proposed. The calibration of those diagrams is far from being obvious. That is why contrarily to other approaches (see, *e.g.*, [8]) we ignore them. Fundamental diagrams are replaced by traffic data which are collected during various situations. Note moreover that our viewpoint, contrarily to other ones like METANET [25], does not necessitate higher order partial differential equations.
- 2. No difficult estimation technique. Implementing any control law requires crucial quantities like the *critical density*, which are time-varying and therefore almost impossible to estimate in real time. They are replaced by *ad hoc* quantities which are deduced at once from easily measurable data.
- 3. **Model-free control.** ALINEA¹ [18, 19, 33, 34] is perhaps the most popular feedback control algorithm for traffic regulation. After first trials in Paris it has been quite often employed in many different places. Following [1], we are using here *model-free control* in the sense of [12, 13] and, more precisely, *intelligent proportional controllers*. This setting, which has been successfully tested in many concrete situations, has already been illustrated via various questions about intelligent transportation systems (see, *e.g.*, [4, 7, 17, 30, 31, 36, 40, 41, 42]). Concrete experiments show today that model-free control not only yields better traffic regulation than ALINEA but is also simpler to implement.²

Some remarks might be useful:

• Let us emphasize that the LWR partial differential equation is only related to simulation purposes and not to traffic control. Such a control would necessitate a real-time calibration, which seems today beyond all reasonable hope.

¹ It is the acronym of Asservissement LINéaire d'Entrée Autoroutière.

² There are many variants of ALINEA in the literature. We have selected for our computer comparisons the version [2].

 Achieving an efficient traffic regulation is simpler than obtaining reliable computer simulations. This ascertainment remains valid in most applications where modelfree control plays a key rôle.

Our paper is organized as follows. Section 2 reviews model-free control and the corresponding *intelligent proportional controller*. Following [3], Section 2.3 investigates the relationship with proportional controllers, which should be considered as the backbone of ALINEA. Modeling issues are discussed in Section 3: The partial differential equation stemming from elementary hydrodynamic conservation laws, some fundamental empirical laws, the space discretization and its use without any fundamental empirical law, the ALINEA and model-free regulation without the critical density. Computer simulations are displayed and commented in Section 4. Section 4.2 in particular exhibits the superiority of model-free control in the presence of sudden changes. Section 5 contains some concluding remarks.

2 Model-free control

2.1 A short review

2.1.1 Ultra-local model

Consider only, for simplicity's sake, SISO (single-input single-output) systems. Elementary functional analysis and differential algebra as used in [12] show as well as practical experiments that most, or at least many, concrete systems may be approximated by the *ultra-local*

$$\dot{y} = F + \alpha u \tag{1}$$

where

- the control and output variables are respectively *u* and *y*,
- the constant $\alpha \in \mathbb{R}$ is chosen by the practitioner such that αu and \dot{y} are of the same magnitude. Therefor α does not need to be precisely estimated.

The following comments might be useful:

- *F* is estimated via the knowledge of the control and output variables *u* and *y*,
- *F* subsumes not only the unknown structure of the system, but also any external disturbance.

2.1.2 Intelligent controllers

Close the loop with the following *intelligent proportional controller*, or *iP*,

$$u = -\frac{F - \dot{y}^* + K_P e}{\alpha} \tag{2}$$

where:

- *y*^{*} is the reference trajectory,
- $e = y y^*$ is the tracking error,
- K_P is a tuning gain.

Combining Equations (1) and (2) yields:

 $\dot{e} + K_P e = 0$

where *F* does not appear anymore. Thus $\lim_{t\to+\infty} e(t) = 0$ iff $K_P > 0$. This local stability property proves that the tuning of K_P is straightforward. This is a major difference with the classic gain tuning for PIs and PIDs (see, *e.g.*, [6] and the references therein).

2.1.3 Estimation of F

Under a weak integrability condition, F in Equation (1) may be "well" approximated by a piecewise constant function F_{est} (see, *e.g.*, [10]). The estimation techniques below are borrowed from [14, 15, 38].³ Let us summarize two types of computations:

1. Rewrite Equation (1) in the operational domain (see, e.g., [43]):

$$sY = \frac{\Phi}{s} + \alpha U + y(0)$$

where Φ is a constant. We get rid of the initial condition y(0) by deriving both sides with respect to *s*:

$$Y + s\frac{dY}{ds} = -\frac{\Phi}{s^2} + \alpha\frac{dU}{ds}$$

Noise attenuation is achieved by multiplying both sides on the left by s^{-2} , since integration with respect to time is a lowpass filter (see [11] for further details). It yields in the time domain the realtime estimate, thanks to the equivalence between $\frac{d}{ds}$ and the multiplication by -t,

$$F_{\text{est}}(t) = -\frac{6}{\tau^3} \int_{t-\tau}^t \left[(\tau - 2\sigma)y(\sigma) + \alpha\sigma(\tau - \sigma)u(\sigma) \right] d\sigma$$
(3)

where $\tau > 0$ might be quite small. This integral may of course be replaced in practice by a classic digital filter.

2. Close the loop with the iP (2). It yields:

$$F_{\text{est}}(t) = \frac{1}{\tau} \left[\int_{t-\tau}^{t} \left(\dot{y}^* - \alpha u - K_P e \right) d\sigma \right]$$

³ They were often used in practice for parameter identification (see, e.g., [27]).

2.2 PI and iP

Consider the classic proportional-integral controller, or PI,

$$u(t) = k_p e(t) + k_i \int e(\tau) d\tau$$
(4)

where $k_p, k_i \in \mathbb{R}$ are constants. A crude sampling of the integral $\int e(\tau) d\tau$ through a Riemann sum $\mathcal{I}(t)$ leads to

$$\int e(\tau)d\tau \simeq \mathcal{I}(t) = \mathcal{I}(t-h) + he(t)$$

where h is the sampling interval. The corresponding discrete form of Equation (4) reads:

$$u(t) = k_p e(t) + k_i \mathcal{I}(t) = k_p e(t) + k_i \mathcal{I}(t-h) + k_i h e(t)$$

Combining the above equation with

$$u(t-h) = k_p e(t-h) + k_i \mathcal{I}(t-h)$$

yields

$$u(t) = u(t - h) + k_p (e(t) - e(t - h)) + k_i he(t)$$
(5)

Replace in Equation (2) *F* by $\dot{y}(t) - \alpha u(t - h)$ and therefore by

$$\frac{y(t) - y(t-h)}{h} - \alpha u(t-h) \tag{6}$$

It yields

$$u(t) = u(t-h) - \frac{e(t) - e(t-h)}{h\alpha} - \frac{K_P}{\alpha} e(t)$$
(7)

Equations (5) and (7) become identical if we set

$$k_p = -\frac{1}{\alpha h}, \quad k_i = -\frac{K_P}{\alpha h} \tag{8}$$

Let us emphasize that this important result, which was first stated in [5, 12], is only valid in discrete time: Formulae (8) become meaningless if $h \downarrow 0$.

2.3 Application to integral controllers

Set in Equation (4) $k_p = 0$. It yields the integral controller, or I,

$$u(t) = k_i \int e(\tau) d\tau \tag{9}$$

Derive both sides in Equation (4):

$$\dot{u} = k_p \dot{e} + k_i e$$

It shows that the PI and I controllers are "close" if \dot{e} remains "small." It implies of course that the initial condition y(0) is "close" to the initial point $y^*(0)$ of the reference trajectory. Let us try to explain why:

- the feedback loop is supposed to render $y y^*$ quickly as small as possible,
- the derivative \dot{e} might therefore be large.

It follows from Equations (5) and (6) that the sampled versions of the iP (2) and the I (9) are "close" (see [3] for more details) if

- the reference trajectory *y*^{*} is "slowly" varying, and starts at the initial condition *y*(0) or, at least, at a point which is quite close to it,
- the disturbances and the corrupting noises are rather mild.

3 Application to ramp metering

3.1 Traffic flow modeling

3.1.1 The LWR partial differential equation

The Lighthill-Whitham-Richards, or LWR, partial differential equation [26, 37], which was derived 65 years ago, is mimicking hydrodynamic conservation laws. It is the simplest model for a macroscopic traffic flow model

$$\frac{\partial}{\partial t}\rho(t,x) + \frac{\partial}{\partial x}f(t,x) = 0$$
(10)

- ρ is the traffic density,
- the traffic flux $f = \rho \times v$ is the product of the traffic density and of the (mean) traffic speed v.

3.1.2 Fundamental laws

There are many empirical laws relating ρ and v. The Greenshield model [16]

$$v(\rho) = v_f \left(1 - \frac{\rho}{\rho_m} \right) \tag{11}$$

and the May formula [29]:

$$v(\rho) = v_f \exp\left(-\frac{1}{a} \left(\frac{\rho}{\rho_c}\right)^a\right) \tag{12}$$

for instance are quite popular. We will not try here to define the corresponding parameters. Note however that they depend on t and x. Their estimation is therefore rather cumbersome.

3.1.3 A space discretization for numerical simulations

Figure 1 displays a portion of a motorway in France, with 7 segments $S\iota$, $\iota = 1, ..., 7$. There are 3 lanes on $S\iota$, $\iota = 1, ..., 6$, and 4 on S_7 . The lengths of the various segments are respectively 4.7, 0.6, 1.4, 1.7, 3.7, 0.6, 0.9 ([km]). Write Q_0 (resp. Q_7) the inlet (resp. output) flow. The traffic flow on $S\iota$ is defined by $(Q_\iota, To_\iota, V_\iota)$, where Q_ι, To_ι and V_ι are respectively the flow rate, the occupation rate and the speed. Our aim is to regulate the flow rates Q_{11} , Q_{12} , Q_{13} on the 3 access ramps in order to ease the traffic flow on the motorway. The control variables are the green light durations GD_κ , $\kappa = 1, 2, 3$, on the 3 on-ramps. The duration of a cycle of the ramp signals is 40s. Set $15s \leq GD_\kappa \leq 29s$.

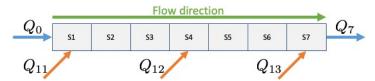


Fig. 1 Segments and ramps of motorway portion

The vehicle conservation principle for each segment ι should be understood as a space discretization of Equation (10). It reads

$$\dot{\rho}_{\iota} = \frac{1}{L_{\iota}} \left(Q_{\iota-1} - Q_{\iota} \right)$$

- ρ_t is the density ([Veh/km]),
- L_t is the length ([km]),
- $Q_{\iota-1}$ is the inlet flow ([Veh/min]): it is the sum of the upstream flow and of the ramp flow if any,
- Q_i is the output flow ([Veh/min]).

Define the *occupancy* rate ([%]) by

$$To_{\iota} = \frac{\rho_{\iota}}{\rho_{\iota,\max}} \times 100$$

where $\rho_{l,\text{max}} = \frac{\lambda_l}{\text{size}}$, λ_l is the number of lanes, size is the mean length ([km]) of a vehicle (here 5.5.10⁻³).

The fundamental diagrams in Figure 2 relate the (mean) speed ([km/h]) and the occupancy, *i.e.*, $v_i = \mathcal{D}_t(To_t)$. They are derived from real data which are, therefore, confidential..

3.2 Feedback loops

3.2.1 ALINEA without calibration

Equation (9) yields ALINEA in ramp metering. For $\kappa = 2$ it reads⁴

$$GD_2 = -K_{I,2} \int_0^t (To_4 - To_4^*) d\tau$$

where the gain $K_{I,2} \in \mathbb{R}$ is set equal to 1. A classic *anti-windup* setting (see, *e.g.*, [6]) is mandatory. The *critical* occupancy rate To_4^* corresponds to the *critical* density ρ_c in Formula (12). A "good" real-time calibration of To_4^* seems today out of reach. We therefore set as in [2]

$$\begin{cases} To_4^*(t+1) = To_4^*(t) + \delta^+ \text{ if } V_4 > V_{4,\text{threshold}} \\ To_4^*(t+1) = To_4^*(t) - \delta^- \text{ if not} \end{cases}$$

where

- V_4 is the mean speed on the 4th segment,
- $V_{4,\text{threshold}} = V_{4,f} 10$,
- $V_{4,f}$ is the *free* speed, *i.e.*, the maximum speed when the traffic is light,
- $\delta^+ = 0.15, \, \delta^- = 0.3.$

3.2.2 iP

Equation (1) reads here

$$To_1 = F_1 + \alpha_1 GD_1$$
$$\dot{T}o_4 = F_2 + \alpha_2 GD_2$$
$$\dot{T}o_7 = F_3 + \alpha_3 GD_3$$

The analogous feedback loops of Equation (2) become

⁴ The cases $\kappa = 1$ and $\kappa = 3$ are similar.

$$GD_{1} = \frac{\dot{To}_{1}^{*} - F_{1} - \mathcal{K}_{1}(To_{1} - To_{1}^{*})}{\alpha_{1}}$$
$$GD_{2} = \frac{\dot{To}_{4}^{*} - F_{2} - \mathcal{K}_{2}(To_{4} - To_{4}^{*})}{\alpha_{2}}$$
$$GD_{3} = \frac{\dot{To}_{7}^{*} - F_{3} - \mathcal{K}_{3}(To_{7} - To_{7}^{*})}{\alpha_{3}}$$

Set $\alpha_i = 30$, $K_{p,i} = 0.5$. To_{κ}^* , $\kappa = 1, 2, 3$ is replaced as above.

4 Simulations

4.1 Generalities

Figures 3 - 13 display convincing results via the setting of Section 3.2.2:

- The reference trajectory is a decreasing time function when the traffic is dense (see Figure 10). This property does not hold anymore if the traffic becomes fluid.
- Figure 11 shows that the green light duration is set to the whole cycle of 40s when the queue on the ramp is large (see Figure 12).
- Figure 13 indicates that the output flow on the ramps are reduced when there is a congestion.

Remark 1 Those results are only slightly better than those obtained via ALINEA with the same calibration of the setpoint. Let us emphasize however that ALINEA when implemented with the critical density ρ_c gives results which are disapointly similar to those obtained without any control, *i.e.*, without ramp metering.

4.2 What is happening with a sudden change?

Assume only 2 lanes, instead of 3, on Segment S_2 : it might be due to an accident or to some work on the highway. Figures 14 shows that the iP, without any new tuning, behave much better than ALINEA, especially outside the congestion hours when ramp metering is more or less useless. Those results agree with Section 2.3.

5 Conclusion

Our approach has already been successfully employed on several French highways. Its adaptative features explains its excellent robustness with respect to unexpected events. Its implementation moreover is quite elementary. There are of course other ramp metering settings which ought to be compared with our viewpoint, especially now those stemming from artificial intelligence. Le us notice that *deep reinforcement learning* methods do not seem today to perform better than ALINEA (see, *e.g.*, [9]).

The *coordination* of several ramp-metering actions on highway networks is being investigated. Coordinated ALINEA techniques have been used in Australia some time ago [35]. It should be clear that any such coordination ought to be connected to a *variable speed limit* (see, *e.g.*, [24]) in order to be more efficient.

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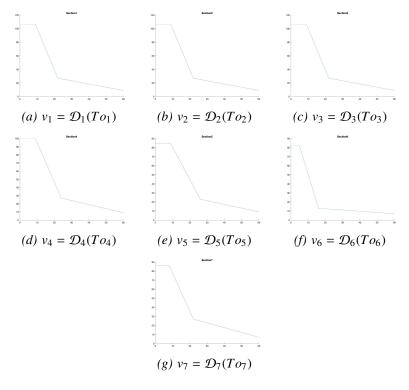


Fig. 2 Diagrams



Fig. 3 Section 1

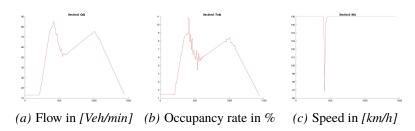
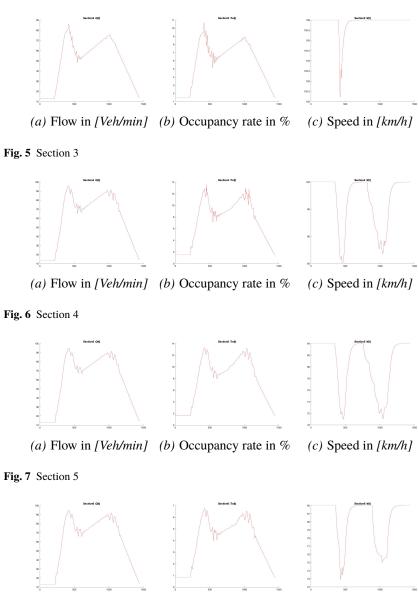
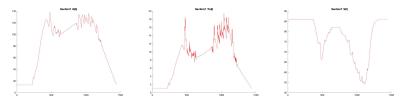


Fig. 4 Section 2



(a) Flow in [Veh/min] (b) Occupancy rate in % (c) Speed in [km/h]

Fig. 8 Section 6



(a) Flow in [Veh/min] (b) Occupancy rate in % (c) Speed in [km/h]

Fig. 9 Section 7

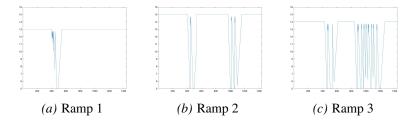


Fig. 10 Setpoint time evolution

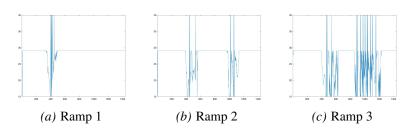


Fig. 11 Time evolution of green lights according to the control inputs GD_1, GD_2, GD_3

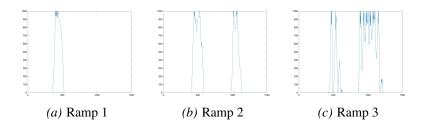


Fig. 12 Queue length in [m]

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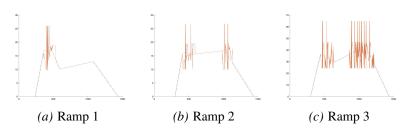


Fig. 13 Demand D_1 , j (- -) and flow Q_1 , j (-) in [Veh/min]

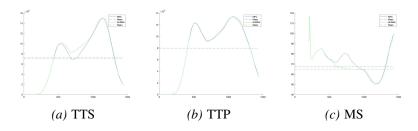


Fig. 14 ALINEA and MFC (TTS: Travel Time Spent, TTD: Total Travel Distance, MS: Mean Speed)